

Statics: Final Exam

2018. 12. 19. 11:00~13:00, Closed Book

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Name: 박민우

Seat No: B1

- 20 1. (20 points) Determine the coordinates of the mass center of the solid homogeneous body formed by revolving the shaded area 90° about the z-axis.

$$dV = \frac{\pi}{4} x^2 dz \quad 2$$

$$V = \int dV = \frac{\pi}{4} \int_0^b a^2 \left(1 - \frac{z^2}{b^2}\right)^2 dz \quad 1$$

$$= \frac{\pi a^2}{4} \int_0^b \left(\frac{z^4}{b^4} - \frac{2z^2}{b^2} + 1 \right) dz = \frac{\pi a^2}{4} \left(\frac{1}{5b^4} z^5 - \frac{2}{3b^2} z^3 + z \right) \Big|_0^b$$

$$= \frac{\pi a^2}{4} \left(\frac{b^5}{5b^4} - \frac{2b^3}{3b^2} + b \right) = \frac{\pi a^2}{4} \left(\frac{1}{5} - \frac{2}{3} + 1 \right) = \frac{2\pi a^2 b}{15} \quad 5$$

$$V \bar{x} = \int x_c dV = \frac{\pi}{4} \int_0^b \frac{4x}{3\pi} a^2 \left(1 - \frac{z^2}{b^2}\right)^2 dz$$

$$= \frac{1}{3} \int_0^b a^3 \left(1 - \frac{z^2}{b^2}\right) \left(\frac{z^4}{b^4} - \frac{2z^2}{b^2} + 1 \right) dz$$

$$= \frac{a^3}{3} \int_0^b \left(-\frac{z^6}{b^6} + \frac{3z^4}{b^4} - \frac{3z^2}{b^2} + 1 \right) dz$$

$$= \frac{a^3}{3} \left[-\frac{z^7}{7b^6} + \frac{3z^5}{5b^4} - \frac{z^3}{b^2} + z \right]_0^b = \frac{a^3 b}{3} \left(-\frac{1}{7} + \frac{3}{5} - 1 + 1 \right)$$

$$i) \quad \bar{x} = \frac{16a^3 b}{105} \times \frac{15}{2\pi a^2 b} = \frac{16a}{14\pi} = \frac{8a}{7\pi} \quad 5$$

$$ii) \quad \bar{y} = \text{Same as } \bar{x}$$

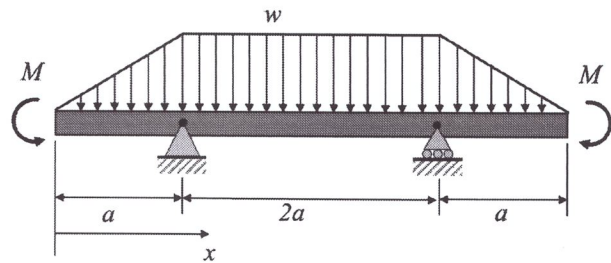
$$V \bar{z} = \int z dV = \frac{\pi a^2}{4} \int_0^b z \left(1 - \frac{z^2}{b^2}\right)^2 dz = \frac{\pi a^2}{4} \int_0^b \left(\frac{z^5}{b^4} - \frac{2z^3}{b^2} + z \right) dz$$

$$= \frac{\pi a^2}{4} \left[\frac{z^6}{6b^4} - \frac{z^4}{2b^2} + \frac{1}{2} z^2 \right]_0^b = \frac{\pi a^2 b^2}{4} \left(\frac{1}{6} - \frac{1}{2} + \frac{1}{2} \right) = \frac{\pi a^2 b^2}{24}$$

$$\bar{z} = \frac{\pi a^2 b^2}{24} \times \frac{15}{2\pi a^2 b} = \frac{5b}{16} \quad 5$$

2. (20 points)

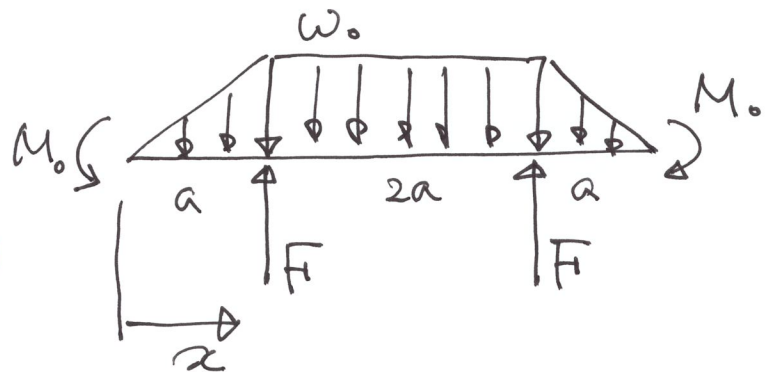
- (a) Draw the shear force and bending moment diagrams for the beam. (Please use $M = wa^2$)
- (b) Find out the moments at $x = a, 2a$, and $3a$.



Free Body Diagram

$$\sum F_y = 0$$

$$\Rightarrow F = \frac{3}{2} \omega_0 a \quad (2)$$

i) $0 < x < a$

$$w = \frac{\omega_0}{a} x$$

$$\frac{dV}{dx} = -w = -\frac{\omega_0}{a} x$$

$$\therefore V = -\frac{\omega_0}{2a} x^2$$

$$\therefore M = -\frac{\omega_0}{6a} x^3 - M_0$$

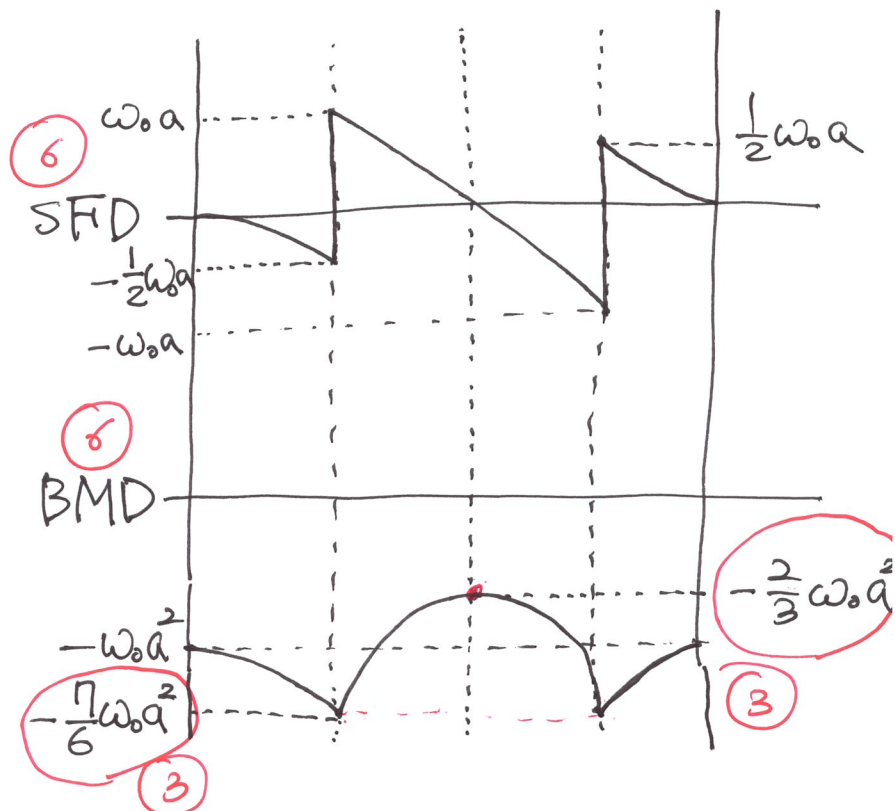
$$(\because \frac{dM}{dx} = V)$$

ii) $a < x < 3a$

$$w = \omega_0$$

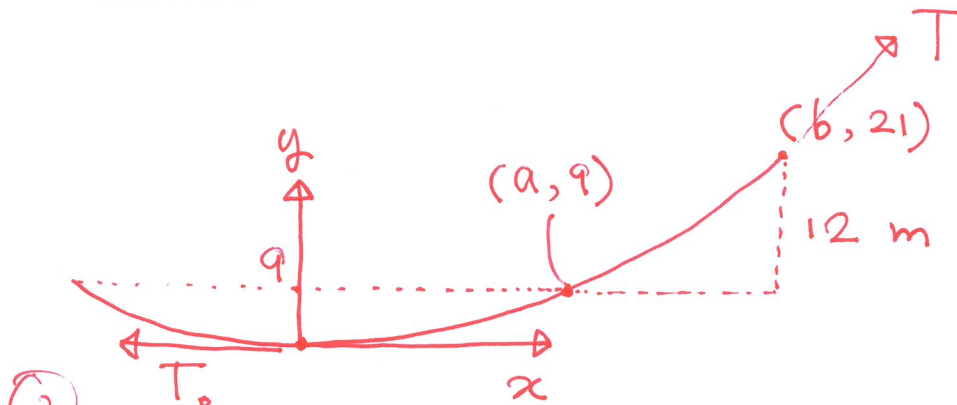
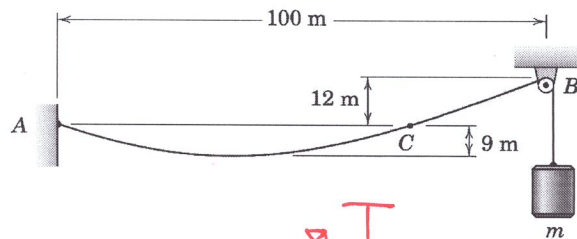
$$V = -\omega_0 x + V_1 = -\omega_0 x + 2\omega_0 a$$

$$M = -\frac{1}{2} \omega_0 x^2 + 2\omega_0 a x + M_1 \quad M_1 = -\frac{16}{6} \omega_0 a^2$$

iii) $3a < x < 4a$: Apply the symmetry!!

3. (20 points) A cable weighing 10 N per meter of length is suspended from point A and passes over the small pulley at B. Because of the small sag-to-span ratio, use the approximation of a parabolic cable.

- (a) Calculate the mass m of the attached cylinder which will produce the sag of 9 m.
 (b) Determine the horizontal distance from A to C.



②

$$a + b = 100 \text{ m}, \quad \omega = 10 \text{ N/m}$$

Governing Eq. $\frac{d^2y}{dx^2} = \frac{\omega}{T_0}$

$$y = \frac{\omega x^2}{2T_0} \quad (3)$$

$$9 = \frac{10}{2T_0} a^2 \rightarrow a = \sqrt{1.8T_0}$$

$$21 = \frac{10}{2T_0} b^2 \rightarrow \underline{b = \sqrt{4.2T_0} = \sqrt{\frac{7}{3}} a}$$

$$a + b = (1 + \sqrt{\frac{7}{3}}) a = 100$$

$$\therefore a = \cancel{100} / (1 + \sqrt{\frac{7}{3}}) \quad \therefore AC = 2a$$

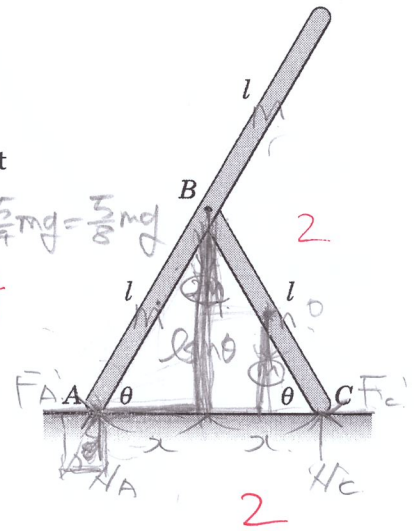
$$\therefore T_0 = \frac{a^2}{1.8} = \frac{100^2}{1.8(1 + \sqrt{\frac{7}{3}})^2} = \frac{200}{1 + \sqrt{\frac{7}{3}}}$$

$$T^2 = T_0^2 + (\omega b)^2$$

$$\therefore m = \frac{T}{g} = \frac{1}{g} \sqrt{T_0^2 + \frac{7}{3} \omega^2 a^2} \quad (5)$$

⑩

4. (20 points) The two uniform slender bars constructed from the same material are freely pinned together at B. The coefficient of static friction at both A and C is $\mu_s = 0.50$. Consider only motion in the vertical plane shown. Determine the minimum angle θ at which slipping does not occur at either contact point A or C.



같이 2당 2점을 m야고 풀기

→ 최소각 θ

$$= \frac{1}{2} \times \frac{5}{4} mg = \frac{5}{8} mg$$

=> 미끄러지지 않는

$$F_A = 0.50 \times N_A$$

$$F_C = 0.50 \times N_C$$

$$= 0.50 \times \frac{11}{4} mg$$

$$= \frac{11}{8} mg$$

$$\sum M_A = 0$$

3

$$2mg \times \frac{1}{2} + mg \times \frac{3}{2} = 2 \times N_C \times \frac{1}{2}$$

$$2N_C = (2m + \frac{3}{2}m)g$$

$$= \frac{7}{2} mg$$

$$\therefore N_C = \frac{7}{4} mg$$

$$\tan \theta = \frac{N_A}{F_A} = \frac{N_A}{0.5 N_A} = 2$$

$$\sum F_y = 0$$

$$2mg + mg = N_A + N_C$$

$$N_A = 3mg - \frac{7}{4} mg$$

$$= \frac{5}{4} mg$$

$$\theta = \tan^{-1}(2)$$

$$\sum M_B = 0$$

회전

3

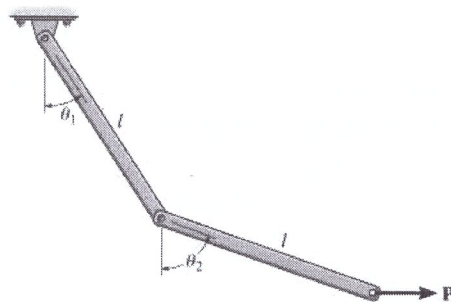
$$\circlearrowleft (N_A) (F_C) (2mg) \quad \circlearrowright (N_C) (F_A)$$

$$\frac{5}{4} mg \times \frac{1}{2} + \frac{7}{8} mg l \sin \theta + mg \times \frac{1}{2} = \frac{7}{4} mg \times \frac{1}{2} + \frac{5}{8} mg l \sin \theta$$

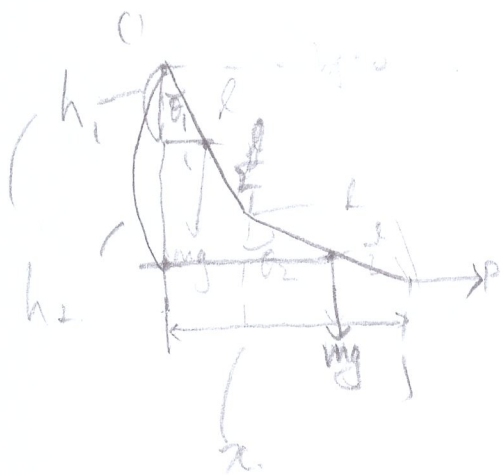
$$\frac{5}{4} \times \frac{1}{2} + \frac{7}{8} l \sin \theta + \frac{1}{2} = \frac{7}{4} \times \frac{1}{2} + \frac{5}{8} l \sin \theta$$

$$\frac{2}{8} l \sin \theta = \frac{14 - 10 - 4}{8}$$

5. (20 points) A horizontal force acts on the end of the link as shown. Determine the angles θ_1 and θ_2 for equilibrium for the two links. Each link is uniform and has a mass m .



Active force diagram = 107



$$\delta W' = 0 \text{ for equilibrium}$$

$$\delta W' = +mg \delta h_1 + mg \delta h_2 + P \delta x = 0$$

$$\text{from } h_1 = \frac{l}{2} \cos \theta_1 \rightarrow \delta h_1 = -\frac{l}{2} \sin \theta_1 \delta \theta_1 \quad 3$$

$$h_2 = l \cos \theta_1 + \frac{l}{2} \cos \theta_2 \rightarrow \delta h_2 = -l \sin \theta_1 \delta \theta_1 - \frac{l}{2} \sin \theta_2 \delta \theta_2 \quad 3$$

$$x = l \sin \theta_1 + l \sin \theta_2 \rightarrow \delta x = l \cos \theta_1 \delta \theta_1 + l \cos \theta_2 \delta \theta_2 \quad 3$$

$$P l (\cos \theta_1 \delta \theta_1 + \cos \theta_2 \delta \theta_2) = mg l \left(\frac{\sin \theta_1}{2} \delta \theta_1 + \sin \theta_1 \delta \theta_1 + \frac{1}{2} \sin \theta_2 \delta \theta_2 \right) \quad 4$$

$$\Rightarrow \delta \theta_1 \times \left(P \cos \theta_1 - \frac{mg}{2} \sin \theta_1 - mg \sin \theta_1 \right) + \delta \theta_2 \left(P \cos \theta_2 - \frac{mg}{2} \sin \theta_2 \right) = 0$$

$$P \cos \theta_2 = \frac{mg}{2} \sin \theta_2 \quad \therefore \tan \theta_2 = \frac{2P}{mg} \quad \theta_2 = \tan^{-1} \left(\frac{2P}{mg} \right) \quad 5$$

$$P \cos \theta_1 = mg \left(\frac{3}{2} \sin \theta_1 \right) \quad \therefore \tan \theta_1 = \frac{2P}{3mg} \quad \theta_1 = \tan^{-1} \left(\frac{2P}{3mg} \right) \quad 5$$